



# Cryocoolers for Space Applications #2

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## Topics

- Space Cryocooler Historical Overview and Applications
- Space Cryogenic Cooling System Design and Sizing
- Space Cryocooler Performance and How It's Measured
- Cryocooler-Specific Application and Integration Example: The AIRS Instrument



# Session 2—Space Cryogenic Cooling System Design and Sizing



## Topics

- **Spacecraft Design and Qualification Requirements Overview**
- **Cryogenic Load Estimation and Management Practices**
- **Estimating Cryocooler Off-State Conduction**
- **Vacuum Level Considerations for Cryogenic Applications**
  - **Gaseous Conduction, Cryopumping, High Emittance Films**
- **Estimating Structural Support Thermal Conduction Loads**
  - **Load Estimating "Rule of Thumb"**
  - **MLI and Gold Plating Lateral Conductivity**
- **Estimating Thermal Radiation Loads**
  - **Radiation Heat Transfer in Cryogenic Applications**
  - **Effect of Material properties and Contaminant Layers**
  - **MLI Performance (Room Temperature vs Cryo)**



# References



- Donabedian, M., “Thermal Uncertainty Margins for Cryogenic Sensor Systems,” AIAA-91-1426, AIAA 26th Thermophysics Conference, June 24-26, 1991, Honolulu, Hawaii, pp. 1-14 (doi: 10.2514/6.1991-1426)
- Gilmore, D.G., “Chapter 19: Thermal Testing,” *Spacecraft Thermal Control Handbook, Volume I: Fundamental Technologies*, The Aerospace Press, El Segundo, CA, pp. 709-725.
- Ross, R.G., Jr., “Requirements for Long-life Mechanical Refrigerators for Space Applications,” *Cryogenics*, Vol.30, No.3, March 1990, pp. 233-238.
- General Environmental Verification Specification for STS & ELV Payloads, Subsystems, and Components, GEVS-SE, Rev A, NASA Goddard Space Flight Center, Greenbelt, MD, 1996, 233 p.
- Ross, R.G., Jr., “Estimation of Thermal Conduction Loads for Structural Supports of Cryogenic Spacecraft Assemblies,” *Cryogenics*, Vol. 44, Issue: 6-8, June - August, 2004, pp. 421-424.



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- **Nast, T.C., “A Review of Multilayer Insulation Theory, Calorimeter Measurements, and Applications,” *Recent Advances in Cryogenic Engineering - 1993*, ASME HTD-Vol. 267, ASME, New York (1993), pp. 29-43. (17 references).**
- **Ross, R.G., Jr., “Chapter 6: Refrigeration Systems for Achieving Cryogenic Temperatures,” *Low Temperature Materials and Mechanisms*, Y. Bar-Cohen (Ed.), CRC Press, Boca Raton, FL (Scheduled to be published in Nov. 2015). (79 references).**
- **[http://www2.jpl.nasa.gov/adv\\_tech/](http://www2.jpl.nasa.gov/adv_tech/) JPL website with 103 JPL cryocooler references as PDFs (R. Ross, webmaster)**



# Principal Cryogenic System Development Challenges



- **5 to 10 YEAR LIFE with  $>0.95$  RELIABILITY**
  - This corresponds to 2,000,000 miles for an automobile with no breakdowns or servicing
  - Also requires compatibility with spacecraft environments and environmental changes over mission life
- **Compatibility with Sophisticated Science Instruments**
  - S/C science instruments demand low levels of vibration and EMI and highly stable temperatures
- **Compatibility with S/C environments and interfaces**
  - Reasonable size and weight
  - Compatible thermal interfaces and heat dissipation levels
  - Compatible electrical interfaces (power level, inrush, ripple current)
  - Compatible with digital communication interfaces



# Cryocooler Technology Drivers



- **5 to 10-year (50,000 hour) operational life mechanical mechanism**
  - Huge potential for wear and mechanical fatigue ( $\sim 10^{10}$  cycles)
- **Sensitive mechanical construction**
  - Precision part fit and alignment
  - Fragile cold-end construction
  - Strong sensitivity to leakage of working fluid (Helium)
- **High sensitivity to contamination**
  - Lubricants or rubbing surfaces generate contaminants (Typically, No lubricants allowed in long-life coolers)
  - Cold surfaces getter contaminants from all sources
- **Complex drive electronics to provide AC waveforms and closed-loop control of piston motions, vibration, and coldtip temperature**
  - AC drive generates vibration, EMI, and high ripple currents
- **Difficult failure analysis**
  - Operation obscured by pressure vessels and vacuum jackets
  - Observation and rework require resealing, decontamination, and refilling — often requiring several weeks



# Programmatic Lessons Learned



- **Simplicity, Maturity and Broad Usage** are Critical to Success
  - **Simplicity** = shorter devel., improved reliability, lower cost
  - **Development level-of-effort** needs to match sponsor/mission time window and funds allocation
  - **Successful technologies** generally funded by multiple sources over many-year time periods before critical maturity reached. **Broad interest base** key to multiple-sponsor continuity
- **Development Time-Constant vs. Mission-Life-Cycle** a Key Issue
  - **Often requirements/need changes** before cryosystem completed
  - **2x change in cryogenic loads** = major redesign
- **Key to Achieving a Successful Space Application**
  - **All S/C requirements fully factored** into R&D phase (launch loads, system interfaces, temperatures, EMI, safety, etc.)
  - **Analytical and test methods** for flight, developed in R&D phase
  - **S/C timeline** matched to cooler development time/maturity level
  - **Stable S/C requirements** to accommodate long cooler devel. time
  - **Simple program interfaces** to allow focus on technical challenges



# Cryocooler R&D Development Process



- ✧ Establish detailed generic cooler requirements for target missions including system operational interfaces, environmental and operational stress levels, reliability, and life
- ✧ Develop preliminary design able to meet requirements
- ✧ Analyze performance and determine principal failure modes and failure-mechanism parameter dependencies
  - Develop and conduct Reliability Physics Analyses
  - Develop and conduct mechanism-specific Characterization and Life Tests of brassboard hardware
- ✧ Resolve or design-out requirement shortfalls
- ✧ Fabricate engineering model
- ✧ Conduct product performance verification tests
  - Full set of Qualification Tests
  - System-level functional tests
  - Multi-year Life Tests
- ✧ Feed back results into next-generation hardware and cooler Specification





# Characterization and Accelerated Life Testing Objectives and Attributes



## OBJECTIVE

- To understand and **quantify the fundamental interdependencies** between performance (failure level), environmental and operational stress level, hardware materials and construction features, and time

## ADVANTAGES

- **Mechanism-level understanding** achieved by selecting specialized tests and facilities targeted at specific degradation stress environments and construction material parameters
- Carefully controlled parameters (generally at parametric levels) with acceleration consistent with accurate extrapolation to use conditions

## LIMITATIONS

- **Expensive and time consuming** — requires specialized testing equipment and modestly long test durations (2 weeks to 5 years)
- Requires multiple tests to address the total spectrum of degradation mechanisms and levels
- Number of specimens insufficient to quantify random failures



# Cryocooler Flight Development Process



- ✧ **Establish detailed mission-specific cooler requirements** including all system operational interfaces, environmental and operational test levels, electronic parts, reliability, and life
- ✧ **Assess heritage design's ability to meet requirements** and modify accordingly
- ✧ **Carefully reevaluate principal failure modes and determine compliance with mission requirements**
  - Reliability Physics Analyses (previously proven techniques)
  - Characterization and Life Tests of flight-like components
- ✧ **Resolve or design-out requirement shortfalls**
- ✧ **Fabricate engineering model and flight units (typically in same build sequence)**
- ✧ **Conduct product performance verification tests**
  - Full set of Qualification Tests
  - System-level functional tests
  - Life Tests often not done (too late, no units, no money)



# Qualification Testing Objectives and Attributes



## OBJECTIVE

- **To rapidly and economically screen hardware designs** and flight articles for prominent (non-wearout) failure mechanisms
- To rapidly assess the relative durability of alternative designs

## ADVANTAGES

- **Quick turnaround — relatively inexpensive**
- Relatively standard procedures allows intercomparison with historical data
- Separate tests (vibe and thermal vac) for important environmental and operational stresses aids identification of high-risk mechanisms

## LIMITATIONS

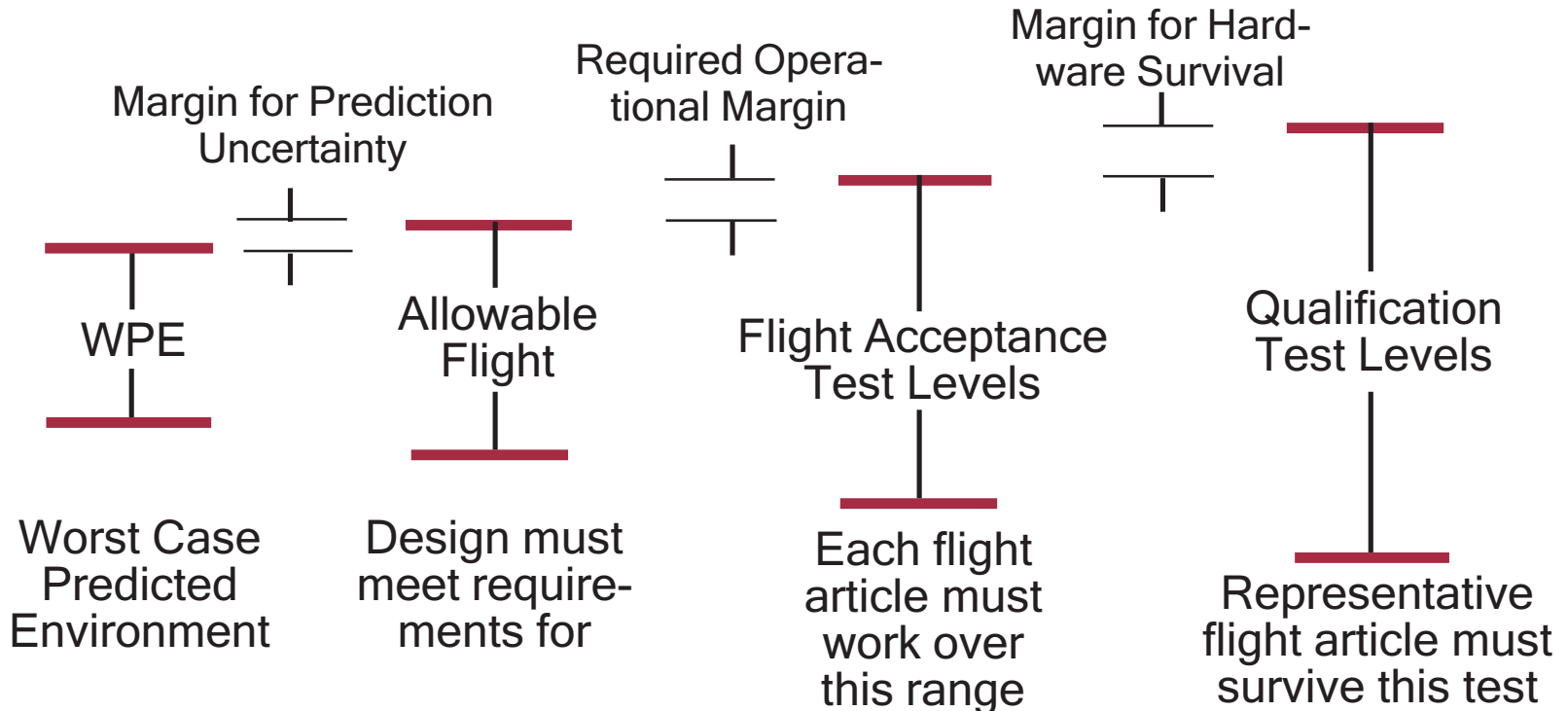
- **Minimal life-prediction capability** (a relative measure of robustness, generally does not quantify life attributes)
- Requires multiple tests and specialized facilities to address the total spectrum of stressing environments
- Number of specimens **insufficient to quantify random failures**



# Typical Space Design and Qualification Requirements



- Aerospace organizations follow a set of institutional requirements for thermal and structural design margins and Qualification test levels.
  - Start with Worstcase Predicted Environments (WPE) throughout the space mission (mission specific)
  - Flight Acceptance (FA), Protoflight and Qualification (Qual) test levels for the hardware are then defined with respect to WPE

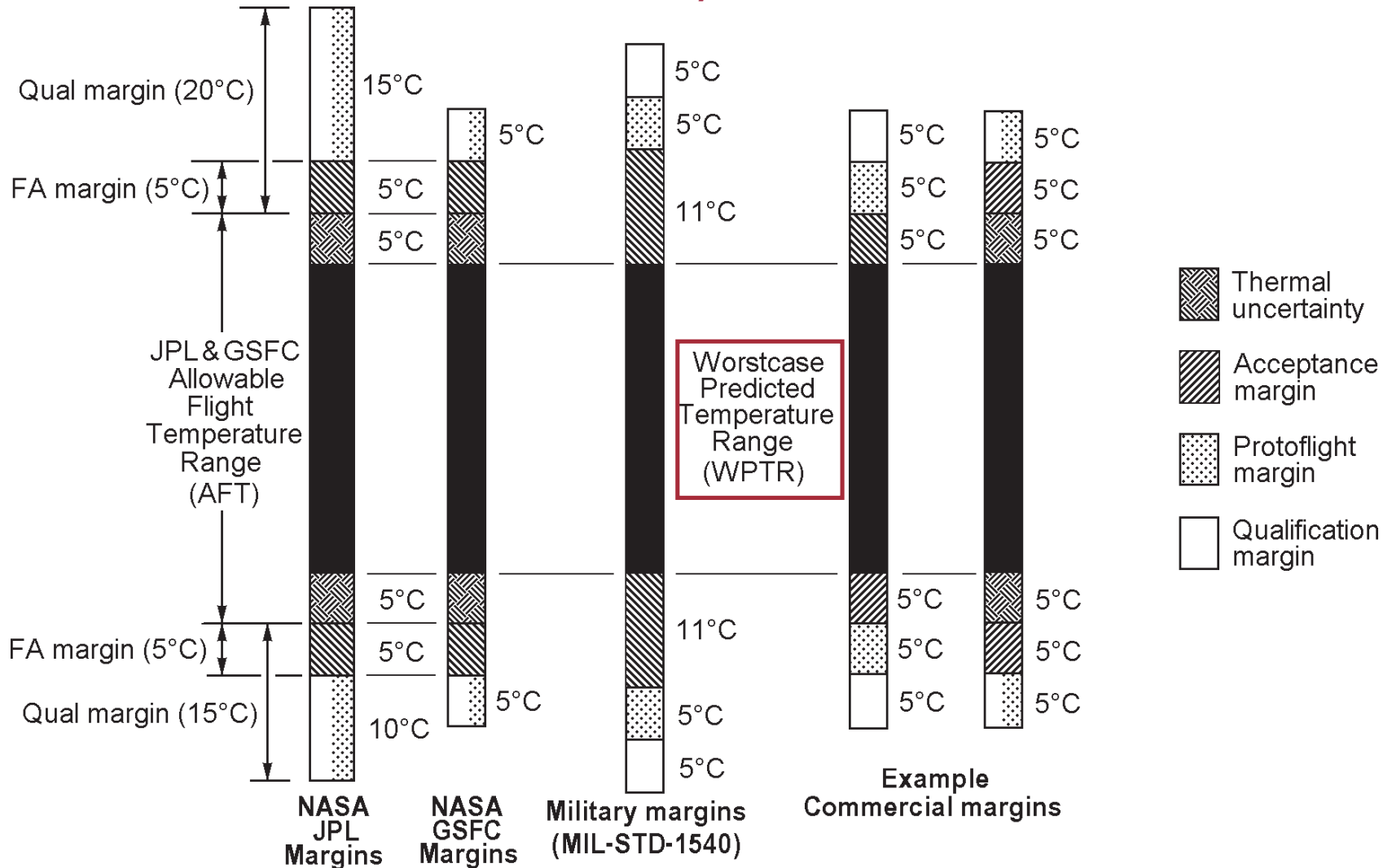




# Typical Space Thermal Design Margin Requirements



*For "Room Temperature" Hardware*





# Full-Up System-Level Testing Objectives and Attributes



## OBJECTIVE

- To accurately assess hardware functionality and reliability with **special emphasis on system synergisms, interactions, and interfaces**

## ADVANTAGES

- Complete system interfaces and operating conditions provides **reliable assessment of subsystem compatibility issues** and degradation mechanisms associated with system interactions or operational stresses
- Inclusion of balance-of-system hardware provides data and confidence in complete functional system

## LIMITATIONS

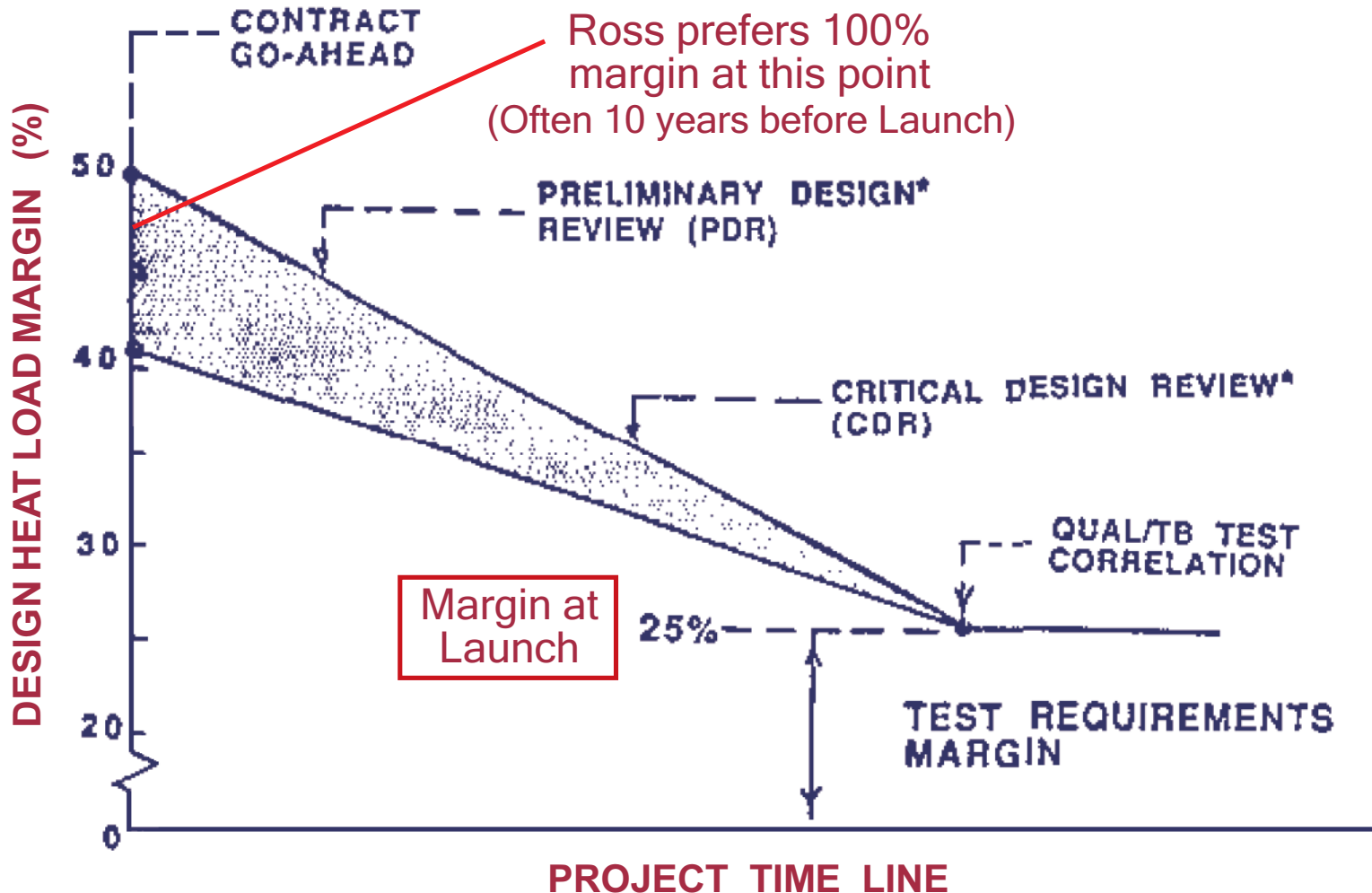
- **Requires complete system with all important balance-of-system components and interfaces**
- Occurs very late in the design cycle; changes at this point are difficult and expensive
- Significant added complexity in constructing and testing complete system



# Recommended Thermal Design/Test Margins for Cryogenic Systems



From Donabedian, M., "Thermal Uncertainty Margins for Cryogenic Sensor Systems," AIAA-91-1426, AIAA 26th Thermophysics Conference, June 24-26, 1991.





# Estimating Cryogenic Loads (The Critical Cryosystem Activity)



- **One of the most important and difficult tasks in cryogenic system design**
  - Needed to select cryocooler design
  - Needed to scope required power and heat dissipation to S/C
  - Needed to identify system thermal design drivers
  - Needed to scope the development risk and cost
- Needs to be accurate to 2x, AND stay within bounds for entire development period (perhaps 10 years)
  - Exceed 2x: generally implies new cooler system design
  - Very difficult to do for an entirely new system w/o prior history
- **Key Steps**
  - Derive a strawman cryogenic system design
  - Estimate the total cooling load over total operating range
  - Acquire performance data for the candidate cryocooler
  - Iterate load projections & cooler selection to get workable design
  - Validate design with detailed calculations and engineering tests





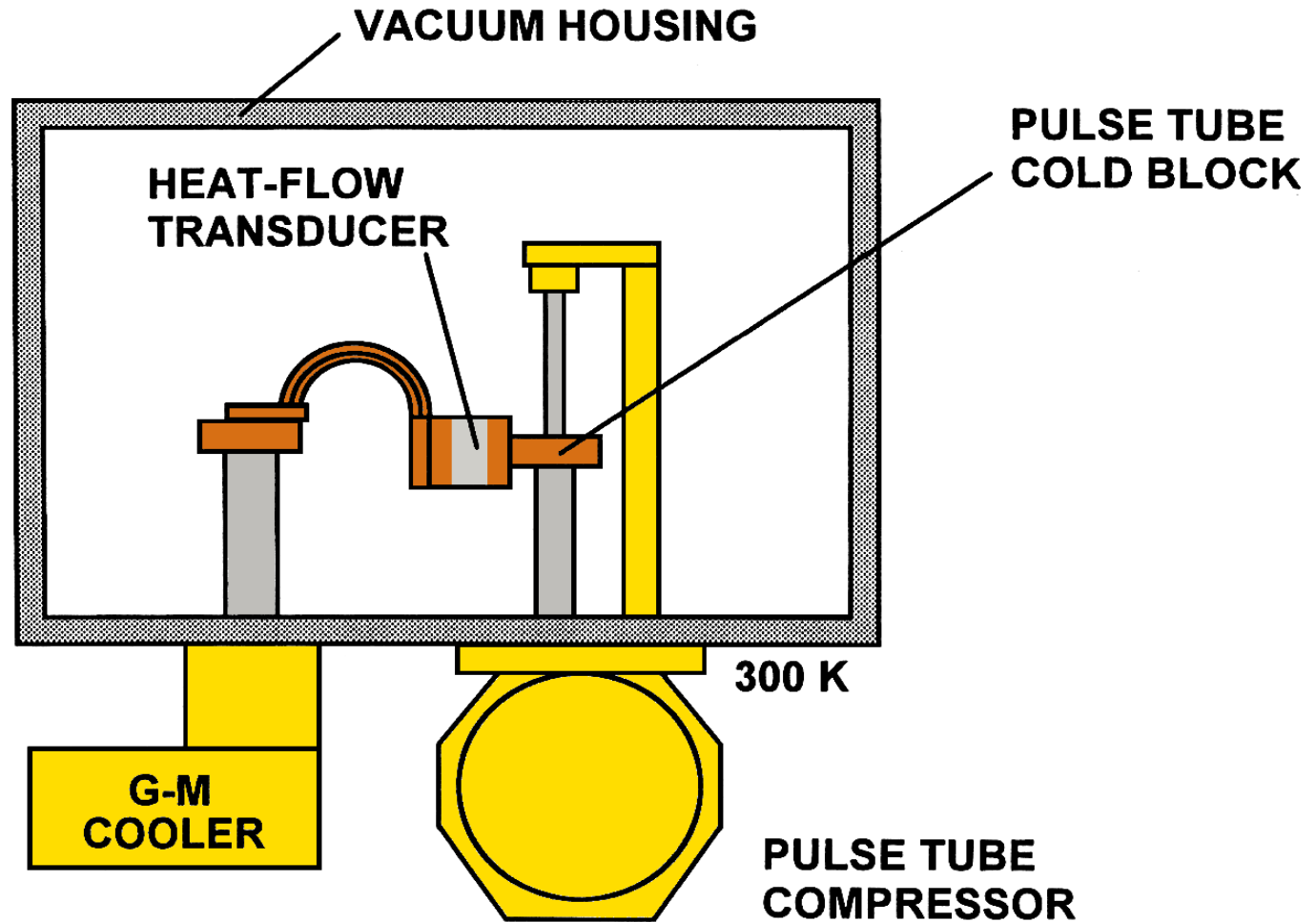
# Principal Space Cryocooler Load Contributors



- **Active Loads**
  - **Direct  $I^2R$  from detectors, motors, electronics, etc**
  - **Cryogenic load (liquefying gases or cooling a fluid or solid)**
- **Parasitic conduction loads of cryosystem interconnections**
  - **Conduction down plumbing and wiring including convection**
  - **Conduction down standby non-operating cryocoolers**
- **Parasitic conduction down cryosystem structural supports**
  - **Conduction down struts and structural members used to support the cryosystem during launch and in space**
  - **Requires structural support concept design**
- **Parasitic radiation from exterior of cryosystem**
  - **Strong function of the surface emittance of application materials**
  - **Strong function ( $T^4$ ) of exterior surface temperatures**
  - **Strongly dependent on surface cleanliness and material purity**
  - **Strongly dependent on MLI construction and compaction**

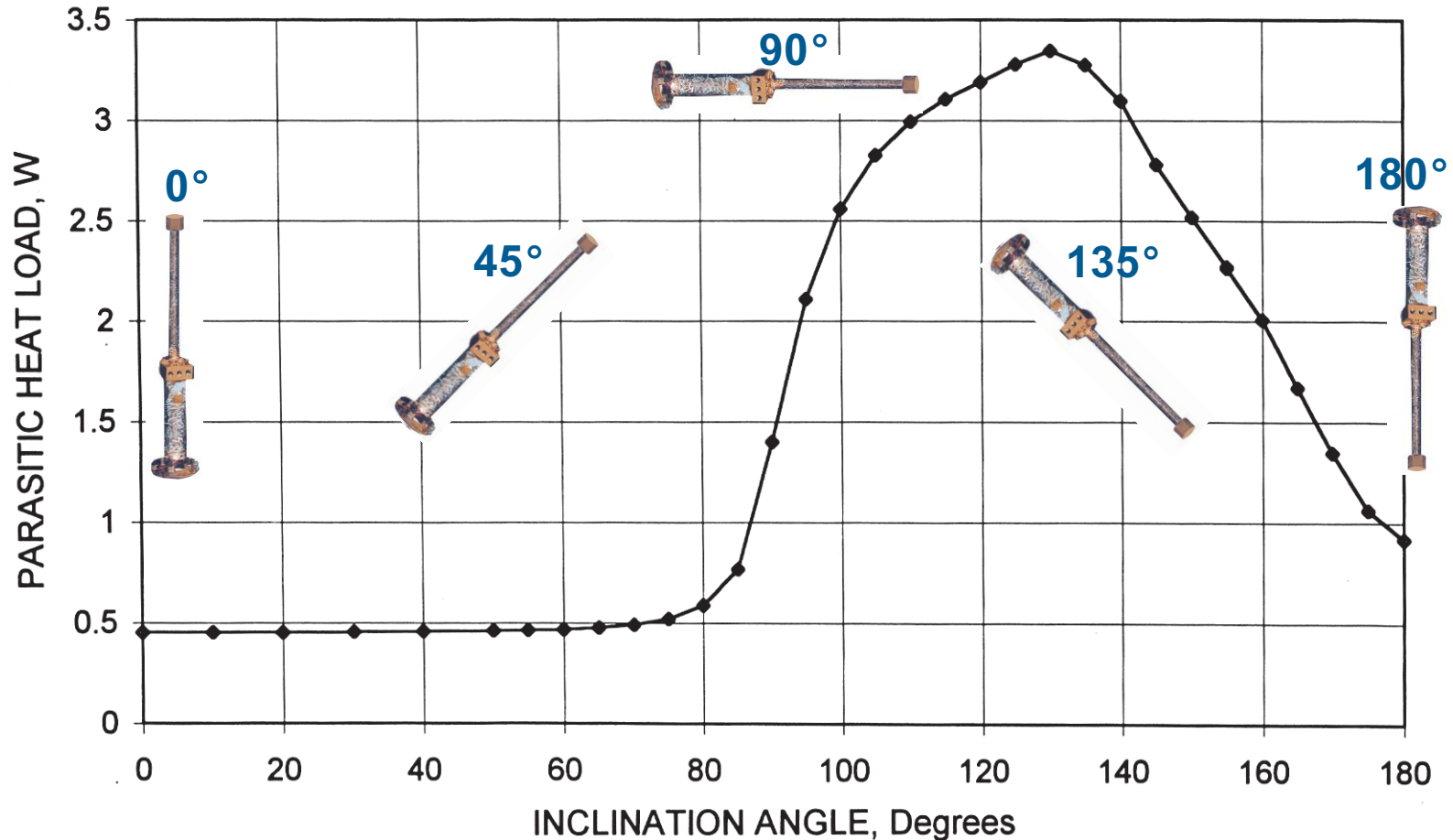


# Cryocooler Off-State Conduction Test Setup



# Coldfinger Off-State Conduction Sensitivity to Inclination Angle

## TRW 6020 PULSE TUBE COOLER

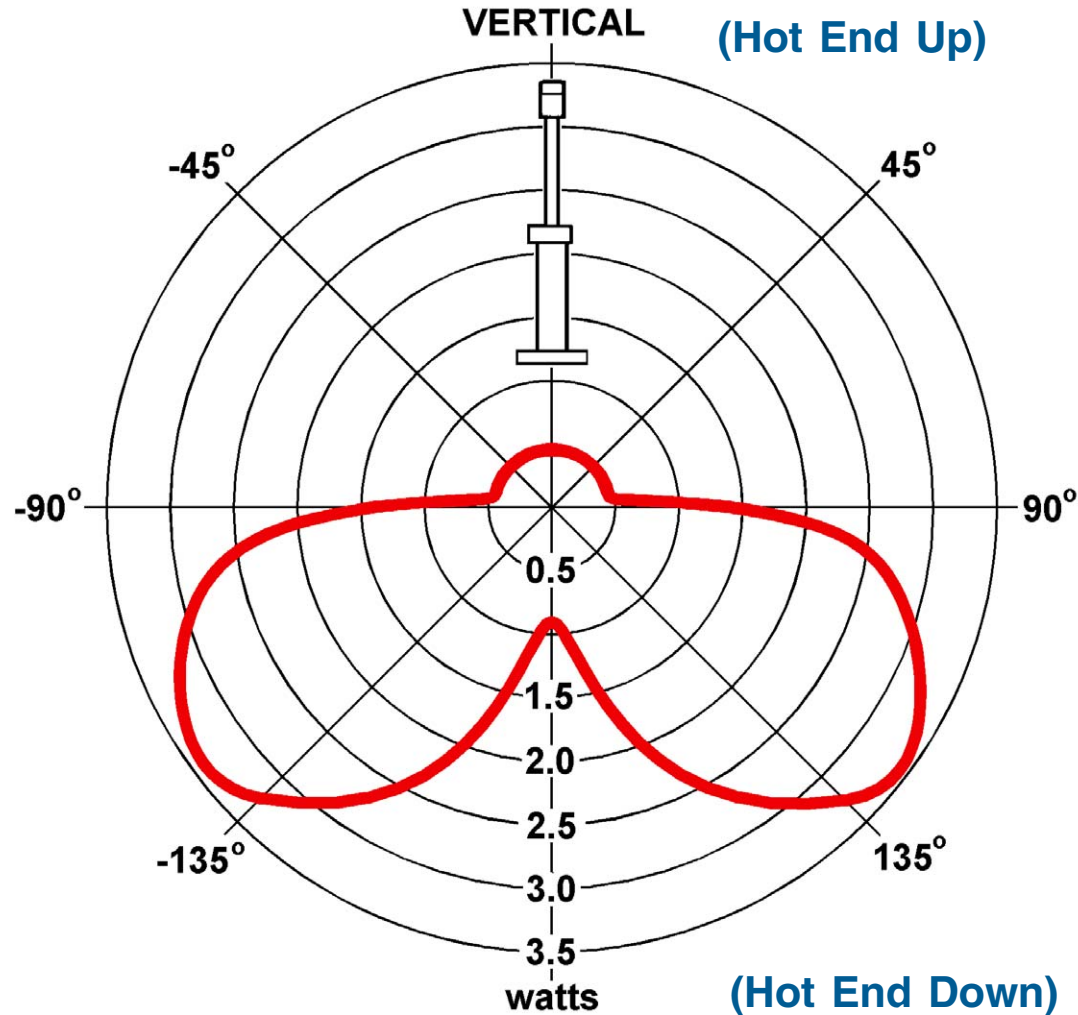




# PT Off-State Conduction at 60K vs Inclination Angle in 1-G Field



## TRW 6020 PULSE TUBE COOLER



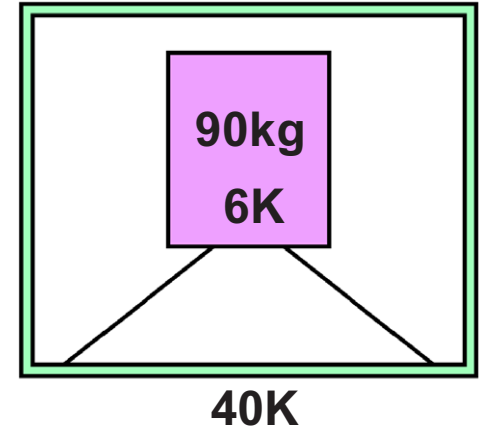


# Estimation of Thermal Conduction Loads for Structural Supports



## OBJECTIVE

- To rapidly and economically estimate structural conduction loads in the early feasibility design phase
- To assess the quality of a structural design against historical benchmarks for achieved conductance



## APPROACH

- Use **scaling equations** built on known relationships between:
  - Material conductivity and temperature
  - Launch acceleration level and assembly mass
  - Support-member cross-sections and launch acceleration level
  - Conductive load and support-member cross-section
- **Scaling Equations calibrated** using a database of successful flight designs.



# Conductive Load Dependences



$$Q = \kappa \Delta T (A / L)$$

where

**Q = Conducted heat (watts)**

**$\kappa$  = Average Material conductivity (watts/cm·K)**

**$\Delta T$  = Differential temperature along member length, K**

**A = Structural member cross-sectional area (cm<sup>2</sup>)**

**L = Structural member length (cm)**

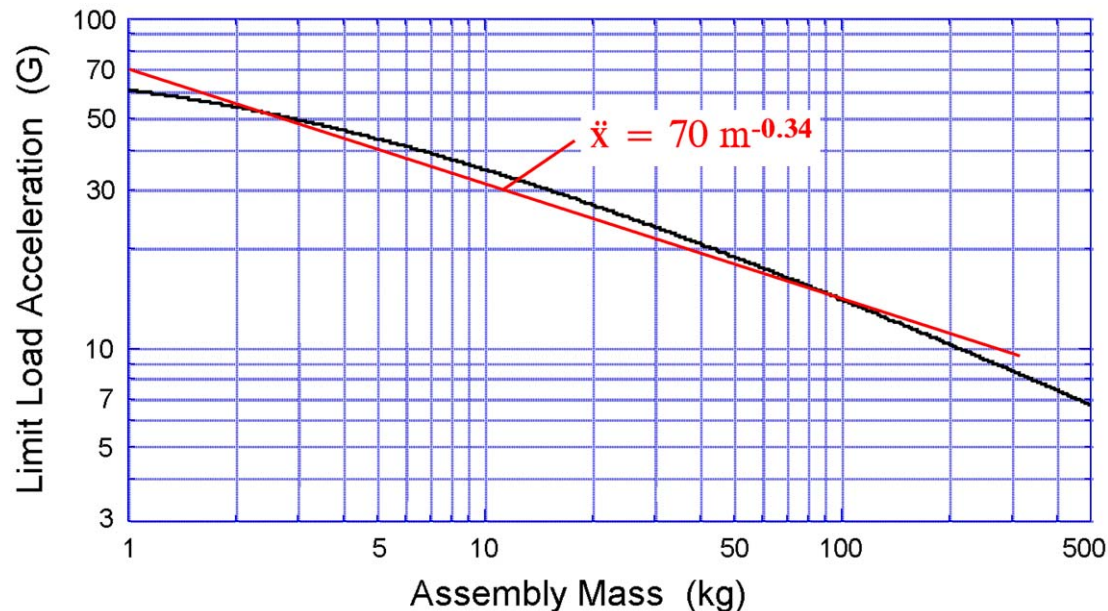
**PROBLEM: Need Estimate for A/L**



# A/L Scaling Dependences



- Stress in support material ( $\sigma$ ) = Force/Area
- For constant material stress: Area must increase  $\propto$  Force
- Force  $\propto$  supported mass ( $m$ )  $\times$  launch acceleration ( $\ddot{x}$ )
- Acceleration ( $\ddot{x}$ ) from Mass Acceleration Curve ( $\ddot{x} \propto m^{-0.34}$ )
- Thus:  $A/L \propto m^{-0.34} \times m$ ; i.e.  $(A/L)_2 = (A/L)_1 \times (m_2/m_1)^{0.66}$





# Overall Scaling Equation for Structural Conductance



Thus:

$$Q_2 = Q_1 \left( \kappa_2 / \kappa_1 \right) \times (m_2 / m_1)^{0.66} \times (\Delta T_2 / \Delta T_1)$$

where

**Q = Conducted heat (watts)**

**$\kappa$  = Average material conductivity (watts/cm·K)**

**m = Supported mass, kg**

**$\Delta T$  = Differential temperature between mass and support point, K**

If we define:

$$\hat{A} = \text{Empirical scaling factor} = Q_1 / (\kappa_1 m_1^{0.66} \Delta T_1) = (A_o / L_o) / m_o^{0.66}$$

Then:

$$Q = \hat{A} \kappa m^{0.66} \Delta T$$

**From Historical Examples:**

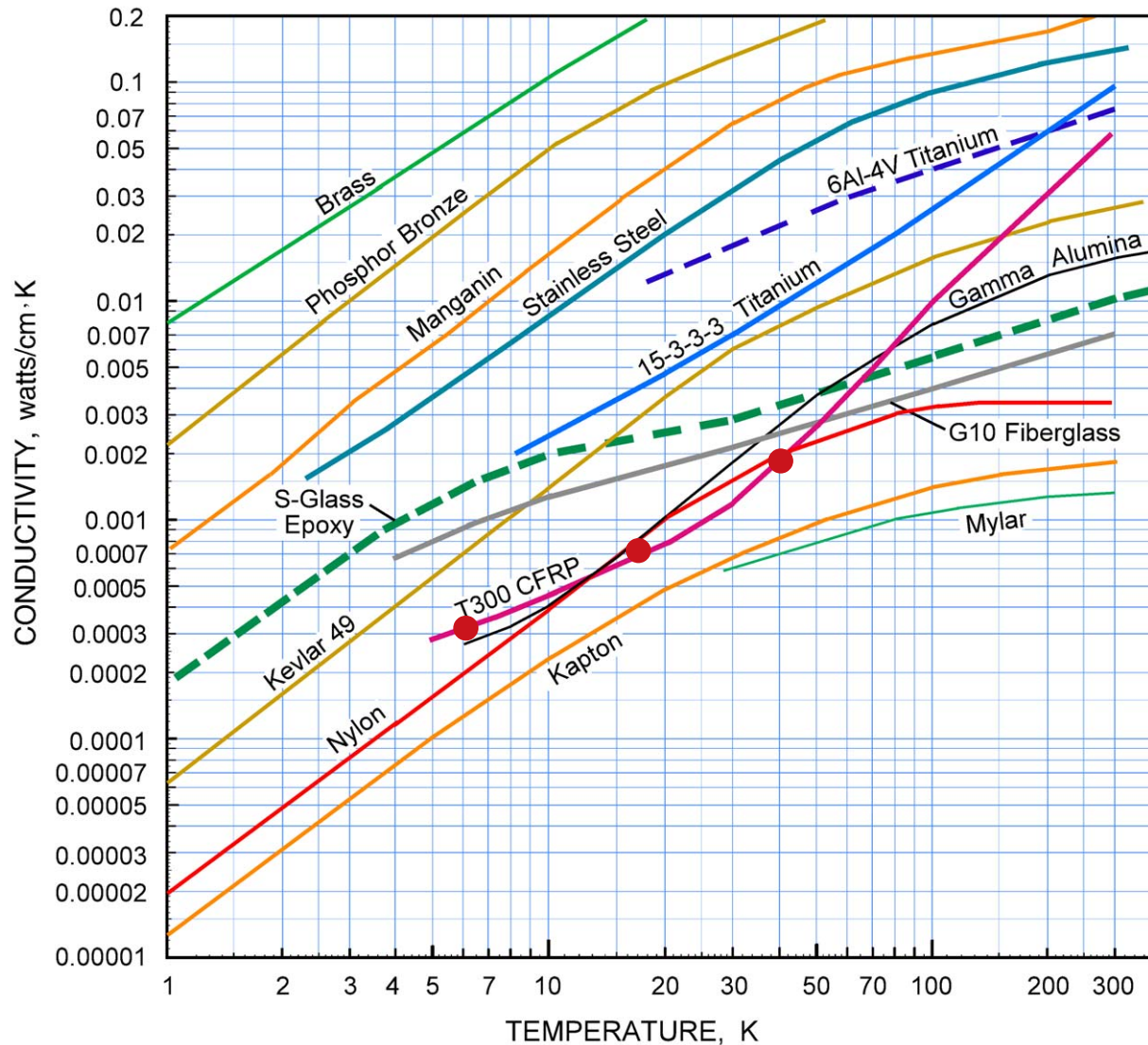
**$\hat{A} \approx 0.28$  for non-optimized (cantilevered) structures**

**$\hat{A} \approx 0.02$  for high-efficiency axially loaded members**



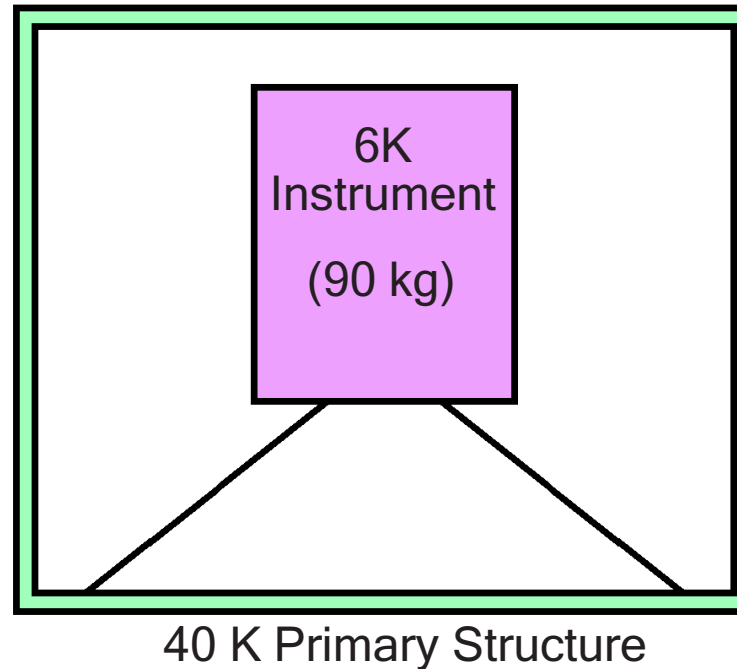


# Thermal Conductivity of Common Low-Conductivity Structural Materials





# Example Space Cryogenic Structure Conduction Estimation Problem



**PROBLEM:** Estimate the structural conduction loads:

$$\begin{aligned} Q &= \hat{A} \kappa m^{0.66} \Delta T \\ &= 0.02 (0.0007)(90)^{0.66} (34) \\ &= 9.3 \text{ mW to } 130 \text{ mW} \\ &\text{(corresponding to } \hat{A} = 0.02 \text{ to } 0.28) \end{aligned}$$



# Watch Out for MLI and Gold Plating Lateral Conductivity

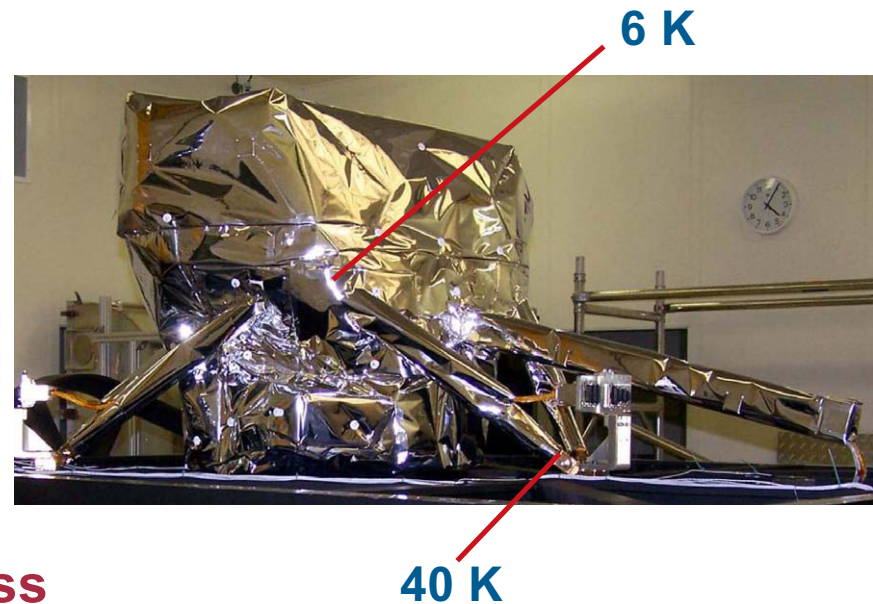


## PROBLEM

- **MLI and Gold Plating have relatively high in-plane conductivity**
- **These materials can create a thermally conductive path between hardware elements with significantly different temperatures**

## LESSONS LEARNED

- **Be very careful about gold plating or wrapping thermally isolating members with MLI**
- **Conductivity of MLI Aluminized layer is about equal to that of 6061-T6 aluminum of equal thickness**





# Vacuum Level Considerations for Cryogenic Applications



## Three Vacuum Level Issues:

- **Gaseous Conduction** from hot surfaces to cold surfaces (Free molecular gaseous heat transfer)
- **Cryopumping heat loads** onto cold surfaces from gases condensing on cold surfaces (heat of fusion added to gaseous conduction)
- **Increased radiation heat loads** on cold surfaces from high emittance condensed gases on cold surfaces

## Typical Vacuum Levels:

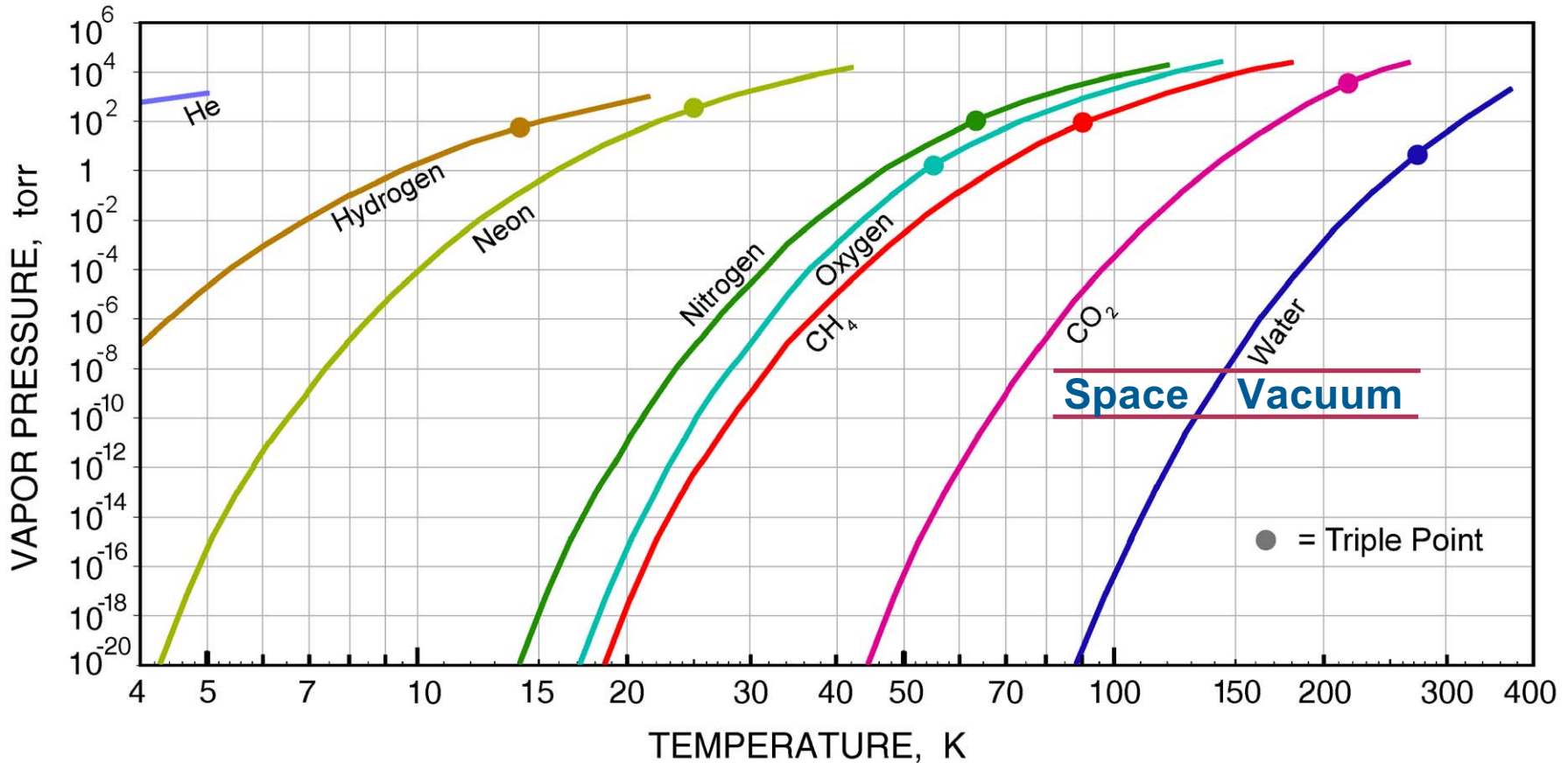
- 10<sup>-4</sup> torr: Run of the mill vacuum chamber
- 10<sup>-4</sup> torr: In space in open Shuttle Bay
- 10<sup>-4</sup> torr: **Inside spacecraft bus** in space (Ross estimate)
- 10<sup>-6</sup> torr: Good quality vacuum chamber
- 10<sup>-8</sup> torr: Inside ultrahigh vacuum chamber
- 10<sup>-8</sup> torr: **Exterior to spacecraft sunlit surfaces** (long term)
- 10<sup>-10</sup> torr: Exterior to spacecraft shaded-side surfaces (long term)



# Vapor Pressures and Condensation Temperatures for Common Gasses



To remain contaminant-free in space requires  $T > 150\text{K}$





# Vacuum Gas Transport and Heat Transfer Considerations



## Key Vacuum Physics Considerations:

- Gas motion in vacuum is free-molecular ... line-of-sight, wall-to-wall with very few gas-gas impacts
  - To pump it, one must intercept the molecules before they reach sensitive cold surfaces
  - Cryopumping with cold shields ( $< 100\text{K}$ ) is highly effective
- From gas transport physics:
  - Rate of  $\text{H}_2\text{O}$  arrival (thickness buildup):  $\dot{\delta}$  ( $\mu\text{m/s}$ ) =  $160 P$  (torr)
  - Cryopumping Heat Transfer Rate:  $\dot{Q}$  ( $\text{W/m}^2$ ) =  $34 P$  (torr)

So, for vacuum pressure levels of water:

Vacuum Level	Time for $1 \mu\text{m H}_2\text{O}$	$\text{H}_2\text{O}$ Cryopumping Heat Transfer
$10^{-4}$ torr	1 minute	34,000 $\text{mW/m}^2$
$10^{-6}$ torr	1.7 hours	340 $\text{mW/m}^2$
$10^{-8}$ torr	7 days	3.4 $\text{mW/m}^2$
$10^{-10}$ torr	2 years	0.034 $\text{mW/m}^2$



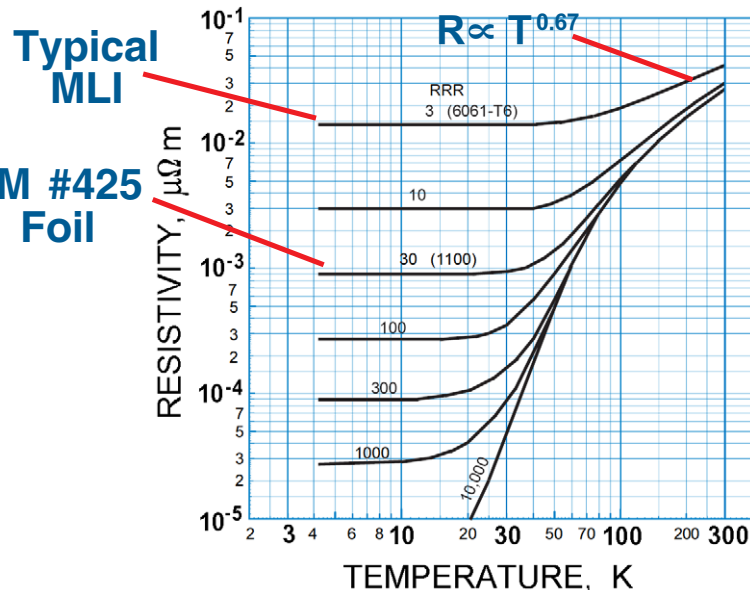
# Radiation Heat Transfer Considerations



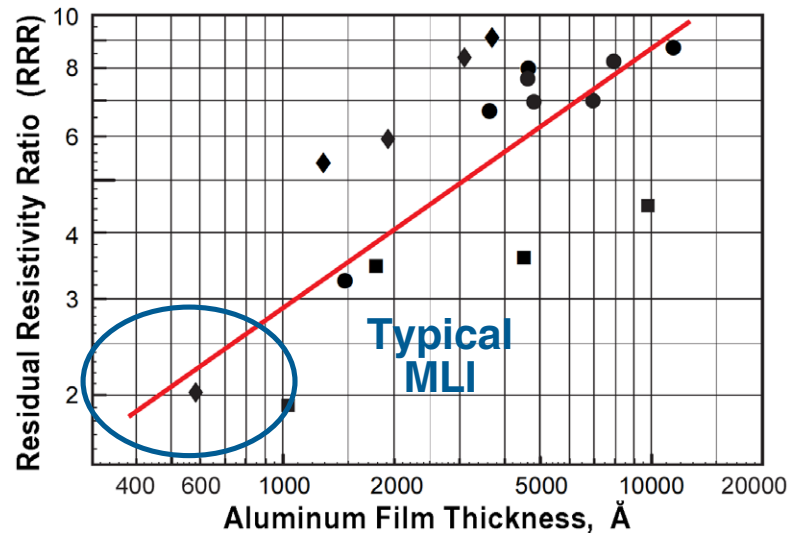
## Key Issues:

- Heat transfer proportional to  $A \epsilon (T_{\text{Hot}}^4 - T_{\text{Cold}}^4) \approx A \epsilon T_{\text{Hot}}^4$
- Emittance ( $\epsilon$ ) (IR absorptance) is dependent upon:
  - Material Surface Electrical Resistance ( $\epsilon \propto R$ )
  - Surface thickness and purity/atomic structure (RRR)
  - Temperature
  - Presence of surface contaminants

### R of Aluminum vs. RRR



### RRR of Aluminum films on MLI

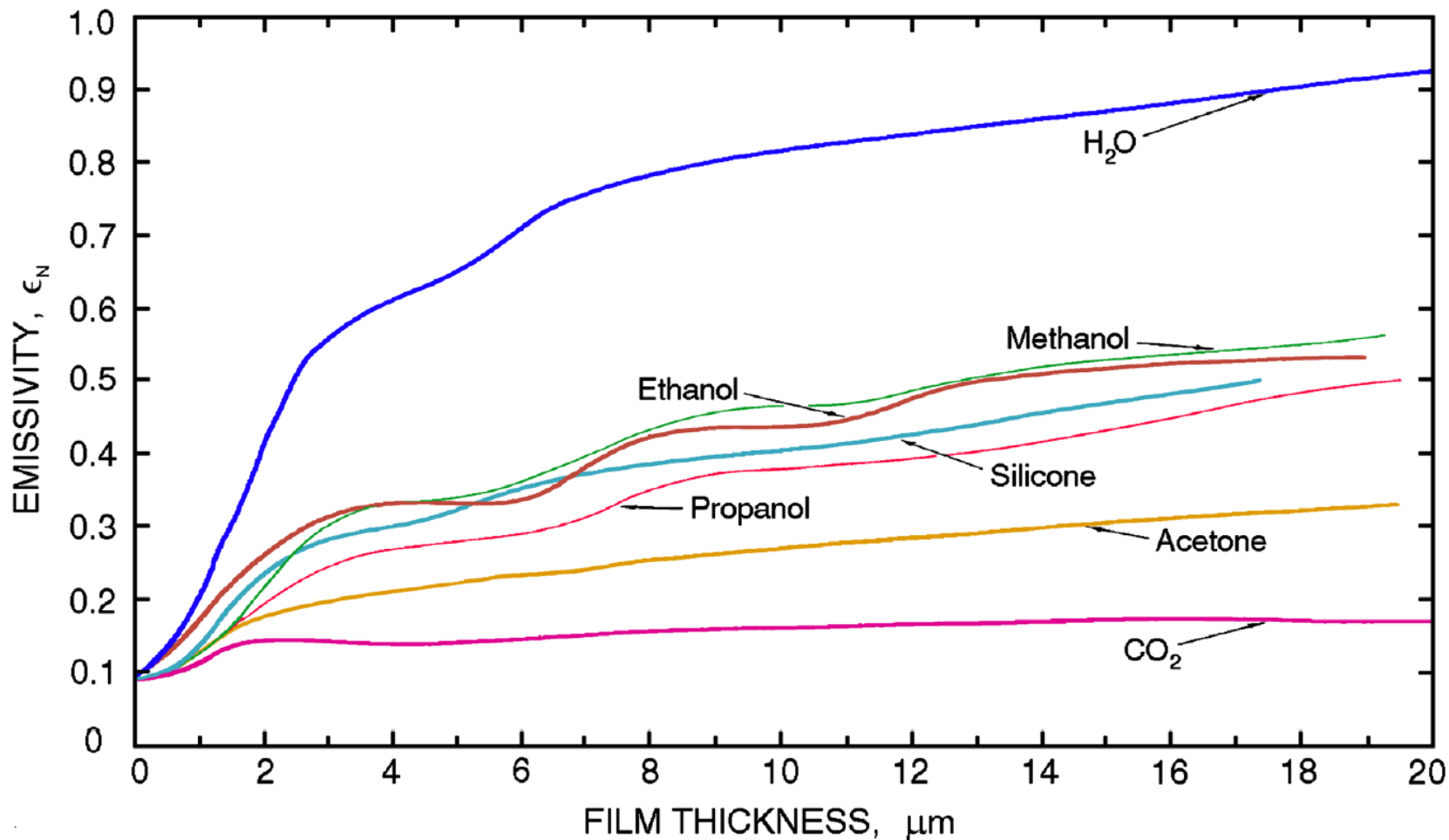




# Emittance Dependence on Contaminant Film Thickness

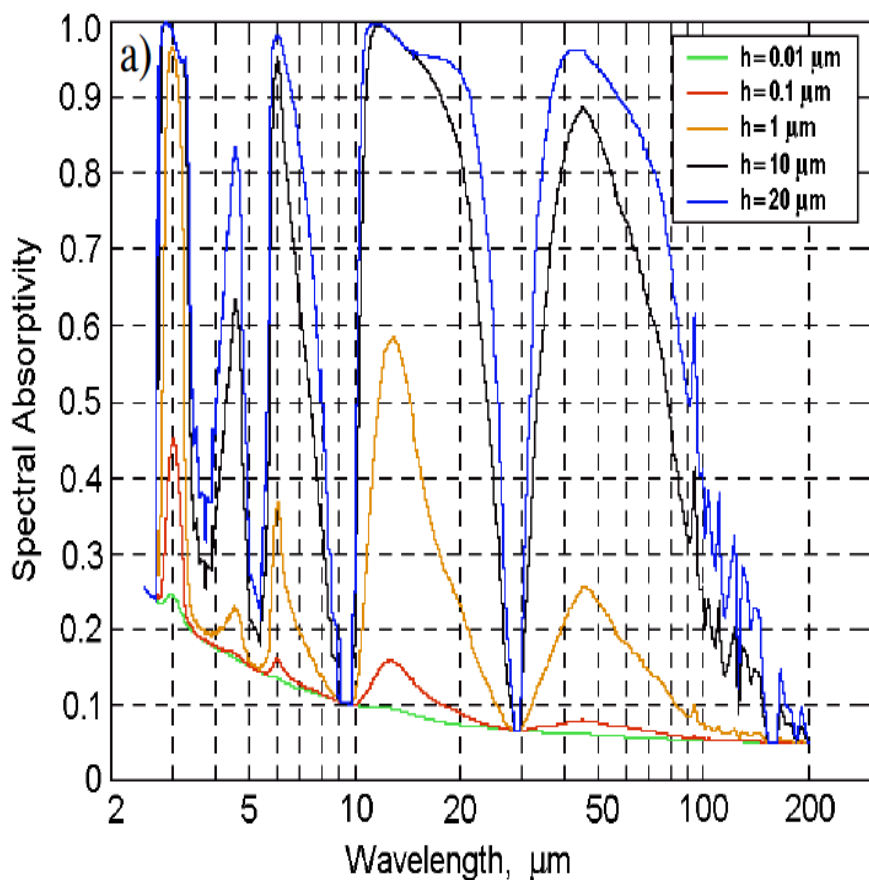


## IR Absorptivity of Contaminant films on Polished Stainless Steel to 300K Blackbody Radiation

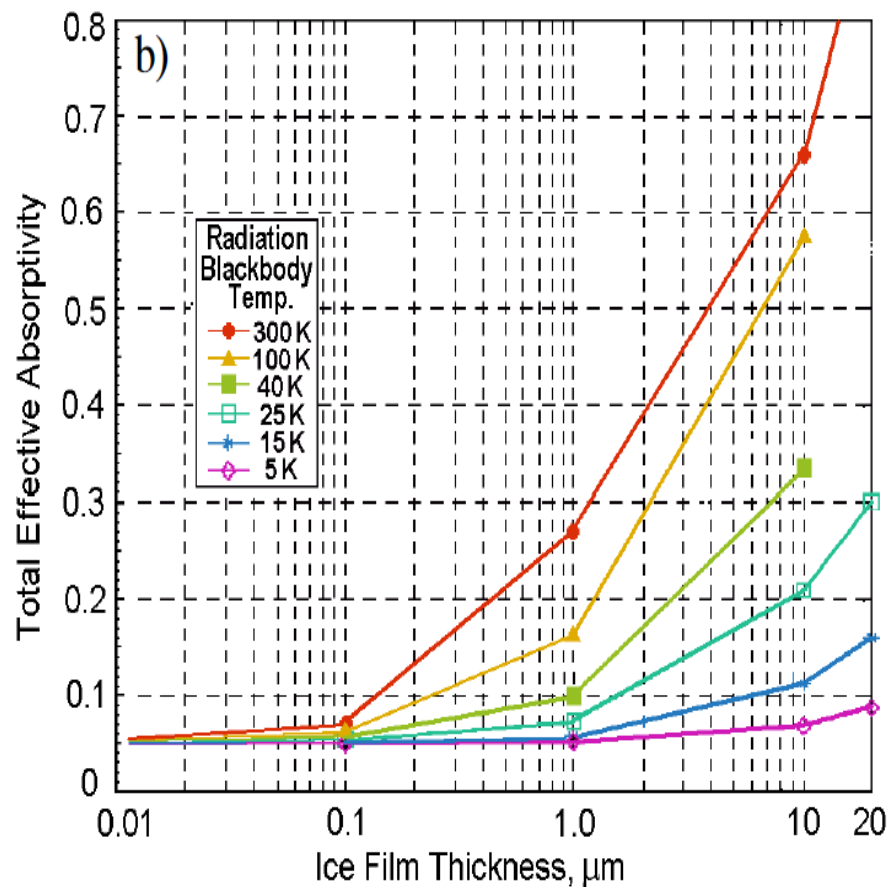




# IR Absorptivity of H<sub>2</sub>O Film (Thickness and Temperature)



- Spectral absorptivity of various thicknesses ( $h$ ) of water ice



- Total IR absorptivity as a function of film thickness for incident radiation from noted blackbody temperatures.



# Estimation of Thermal Radiation Loads with Conventional MLI



## Classic Lockheed MLI Equation

$$q = q_c + q_r = \frac{\text{Conduction } C_c N^{2.56} T_m}{n} (T_h - T_c) + \frac{\text{Radiation } C_r \epsilon_o}{n} (T_h^{4.67} - T_c^{4.67})$$

### where

$q$  = total heat flux transmitted through the MLI (mW/m<sup>2</sup>)

$q_c$  = conductive heat flux transmitted through the MLI (mW/m<sup>2</sup>)

$q_r$  = radiative heat flux transmitted through the MLI (mW/m<sup>2</sup>)

$C_c$  = conduction constant =  $8.95 \times 10^{-5}$

$C_r$  = radiation constant =  $5.39 \times 10^{-7}$

$T_h$  = hot side temperature (K)

$T_c$  = cold side temperature (K)

$T_m$  = mean MLI temperature (K); typically  $(T_h + T_c)/2$

$\epsilon_o$  = MLI shield-layer emissivity at 300 K = 0.031

$N$  = MLI layer density (layers/cm)

$n$  = number of facing pairs of low-emittance surfaces in the MLI system



# Estimation of Thermal Radiation Loads with Cryo MLI

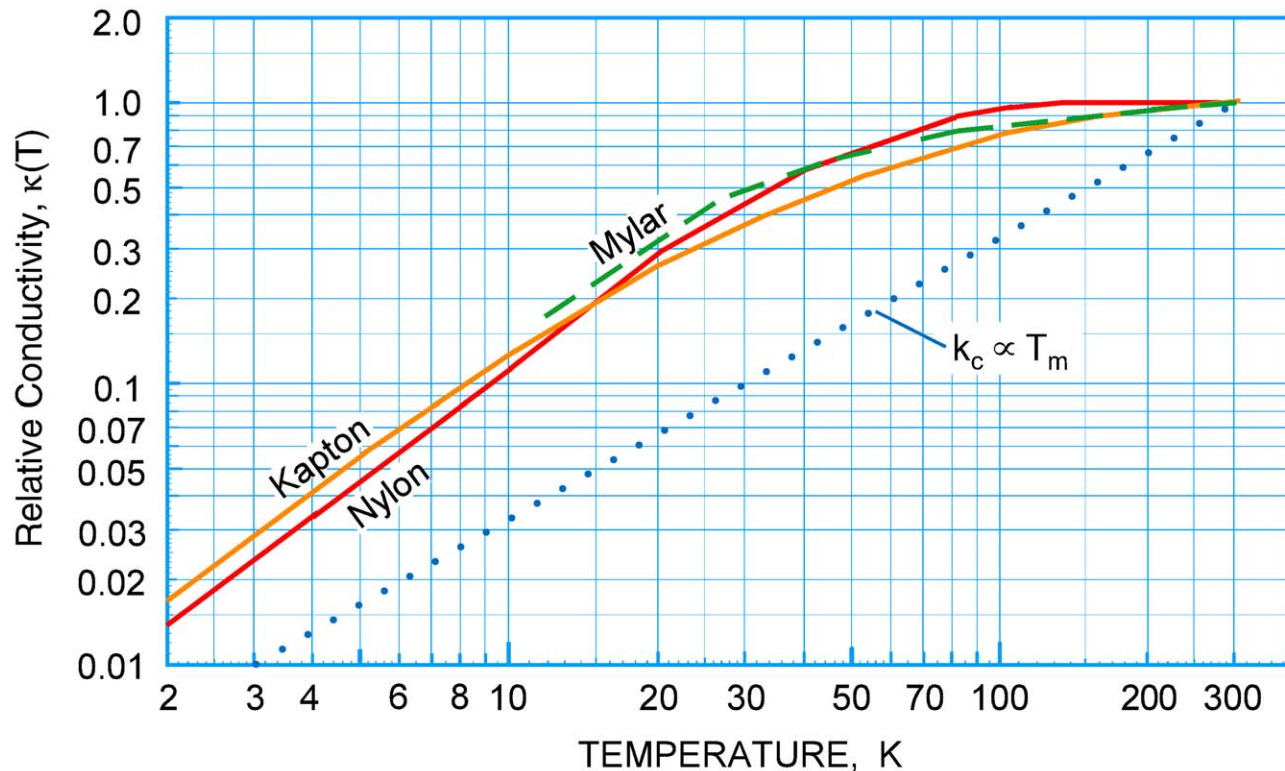


## Modified Lockheed MLI Equation

Conduction

Radiation

$$q = q_c + q_r = \frac{k_o \kappa(T)}{n} (T_h - T_c) + \frac{C_r \epsilon_o}{n} (T_h^{4.67} - T_c^{4.67})$$

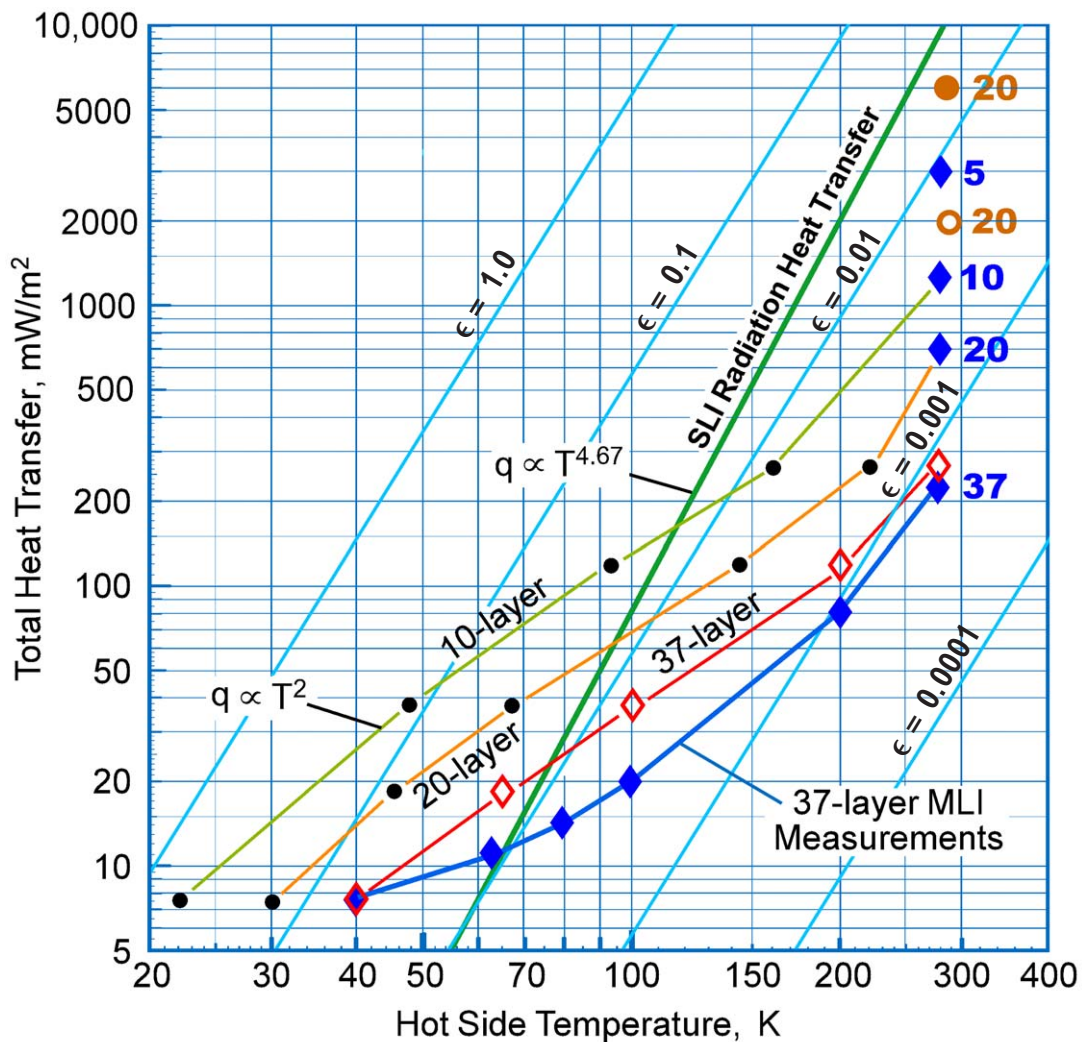




# Measured Thermal Radiation Loads with Room-Temperature MLI



As a function of Hot Side Temperature



## Key:

- SLI Radiation Absorbed ( $\epsilon_H = 1, \epsilon_c = 6.8 \times 10^{-4} T_H^{0.67}$ )
- 20 JPL 20-layer Cassini (SSAK+5EK+15MN+AK)
- 20 JPL Duo-layer Cassini (SSAK+5EK/15MN+A) (20 layers in 2 blankets with staggered seams)
- ◆ 10 Unperf. DAM 1-SN MLI (X = number of layers) (LMSC dewar minimum achiev. layer density)
- ◇ Modeled results for LMSC 37-layer DAM 1-SN
- Modeled results for LMSC 20-layer DAM 1-SN
- Modeled results for LMSC 10-layer DAM 1-SN
- Lines of constant Effective Emittance

## Bottom Line:

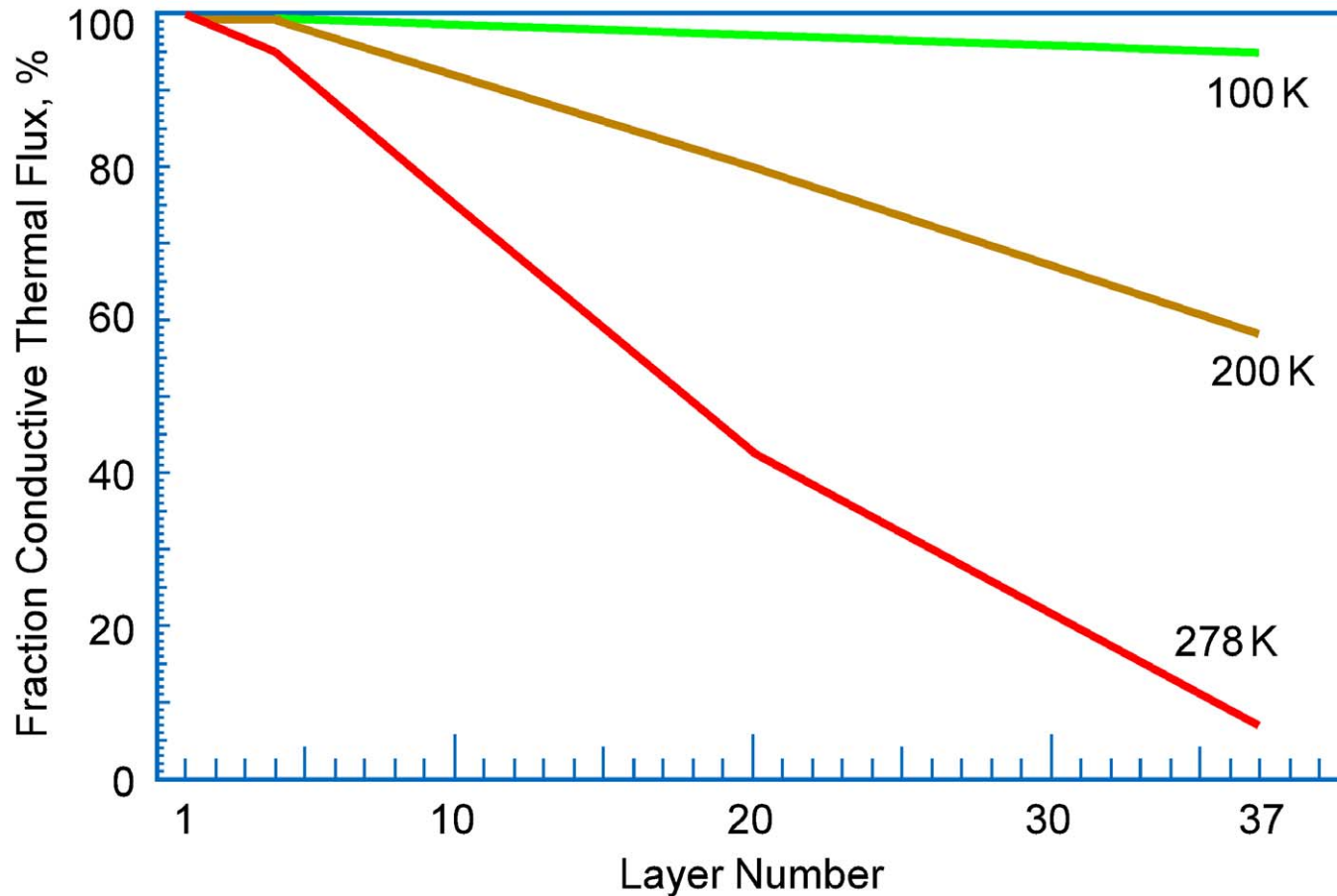
- Room-temperature MLI quickly degrades at lower Hot-Side Temps. Avoid using at  $T_H < 100K$
- Spacecraft MLI 10x higher emittance than Dewar MLI



# Relative Role of MLI Conductance as a function of Hot Side Temperature



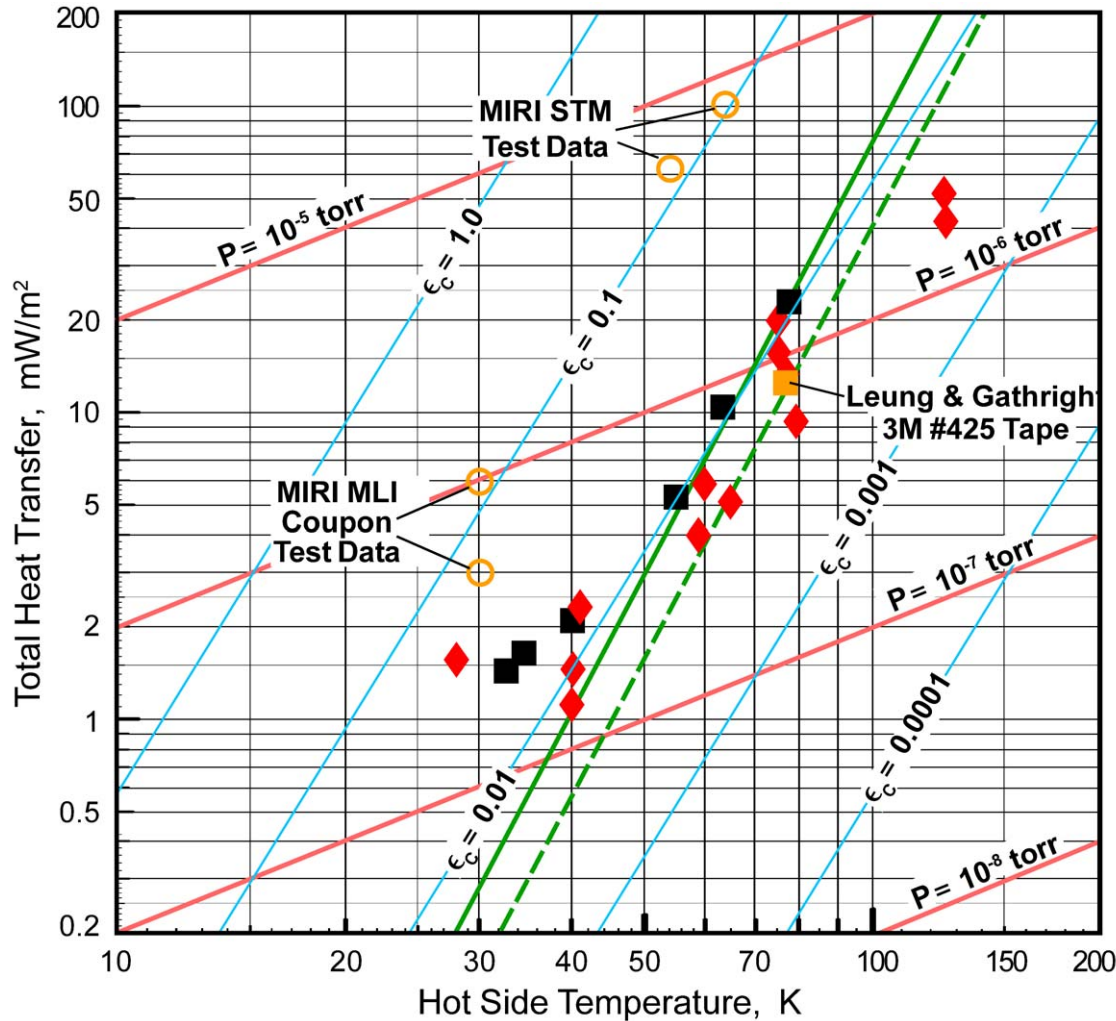
Lockheed 37-layer Dewar MLI ( $k_0 = 25$ )



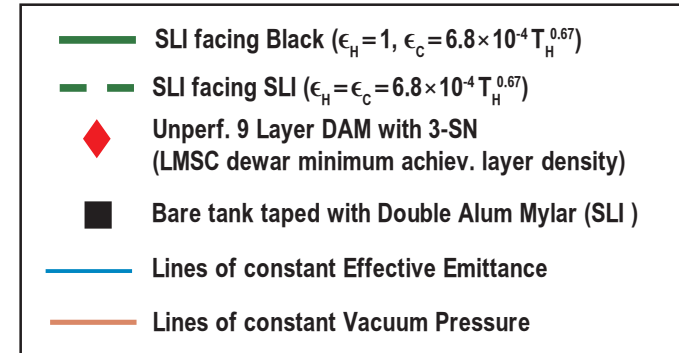
Hotside  
Temp



# Measured Thermal Radiation Loads with Cryo-MLI & SLI



## Key:



## Bottom Line:

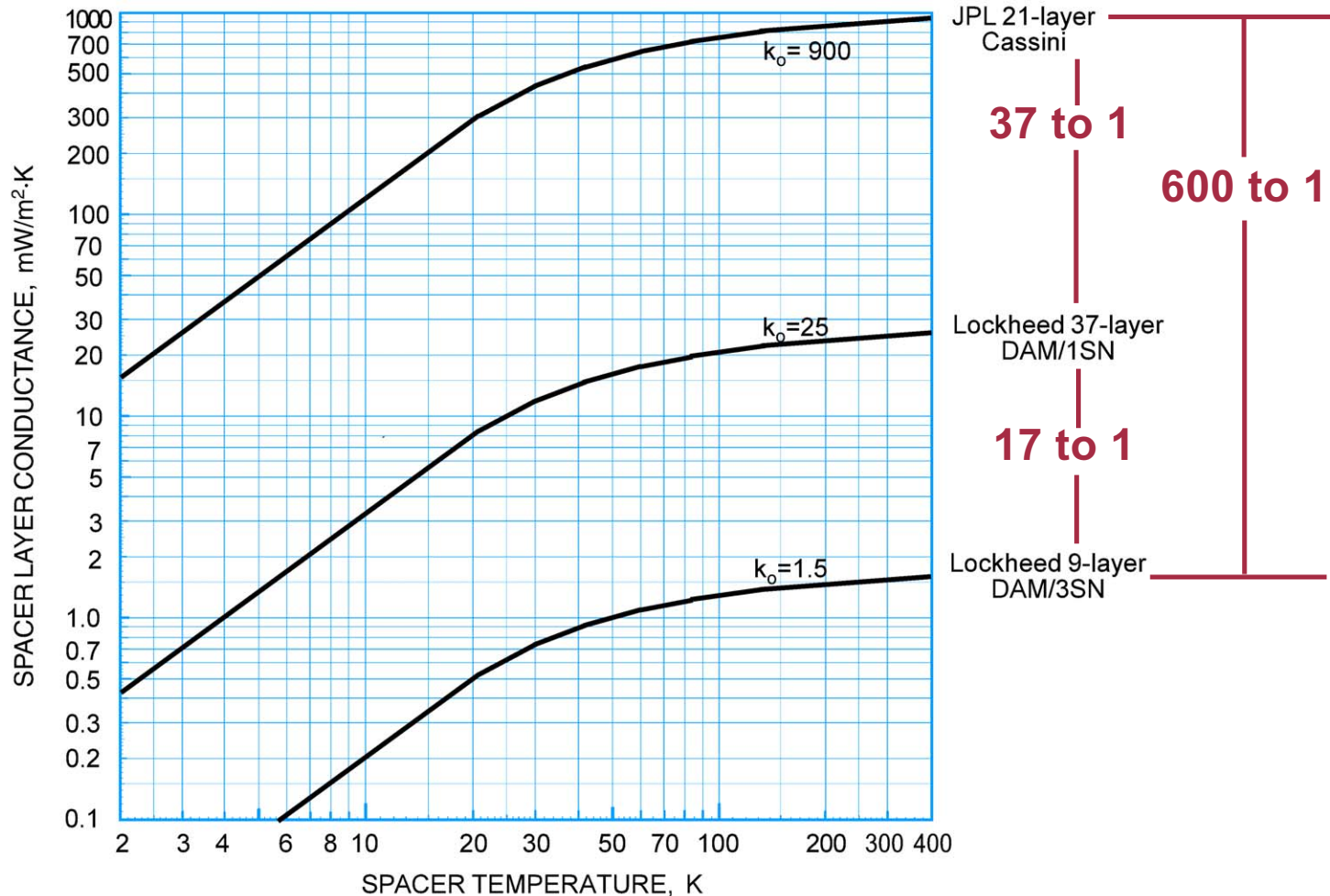
- Cryo Dewar MLI can improve upon SLI emittance down to 40 K Hot-Side Temps (but only by 2x)
- Spacecraft MLI has no hope at cryogenic Hot Side Temps
- 3M #425 tape is comparable to Cryo MLI
- Gas conductance seen to impact heat transfer for  $T_H < 50$  K



# Measured Conductances of Various MLI Constructions of Various MLI Constructions



## 600 to 1 Variability in MLI Conductance between Cryo-dewar MLI and S/C MLI





# System Design and Sizing Summary



- **Designing cryogenic systems for space (or for ground) is a complex process requiring careful management**
  - **Accurate early identification of system requirements**
  - **Conservative margins applied for inevitable changes associated with improved design fidelity**
  - **Systematic Characterization & Qualification of system to burn-down margins and reduce risk**
- **Cryogenic system designs typically have LARGE uncertainties**
  - **Structural conduction loads**
  - **Vacuum level (gaseous conduction & cryopumping)**
  - **Emittances (surface material properties & contaminant levels)**
  - **MLI effective emittance (conductance, unintended contact)**